# DUAL PHASE DAMPING IN SIMPLE SHOCK MOUNTS

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Received on June 19, 1974 and in revised form on September 10, 1974

#### Abstract

Response of a simple mounted system to time varying transients like rounded step and pulse displacements applied at the foundation is investigated. The system employs a dual-phase damping characteristic defined by a set of four parameters. Variations of the peak acceleration and displacement with these four parameters are computed. Optimum values of the parameters at which the minimum peak acceleration or minimum peak displacement occurs are then sought. The main advantage in using a dual-phase damping over single phase constant damping is the facility of minimising the peak acceleration or displacement.

Key words: Shock isolation; Damping.

# 1. INTRODUCTION

This paper presents an investigation of the response of a simple mounted system (Fig. 1. 1 a and b) having nonlinear dual-phase damping to a transient time varying input impressed on the foundation. The peak acceleration and displacement of the mass M are particularly considered here in bringing out the variations of their respective magnitudes with different damping parameters of the dual-phase damping. Optimum values of the parameters are found in order to achieve either minimum acceleration or displacement, one subject to the constraint of the other.

The purpose of the shock absorber is to reduce the acceleration and displacement of the mass M. The components of M are less affected if they experience less acceleration and displacement. This paper considers the effect of the dual-phase damping instead of constant damping on the peak acceleration and displacement, as the effect of the former is found to be better than that of constant damping.

The damping system may be represented by a linear spring and a nonlinear dashpot (Fig. 1. 1 a and b) along with the dual-phase damping

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characteristic defined by the four parameters  $\zeta_1$ ,  $\zeta_2$ , A and B. Realisation of the dual-phase damping characteristics is evident in the sketches of Fig. 1.2 where a spring-loaded ball or a return valve adapt to the changes in damping force offering the desired characteristics [1]. The four parameters could be changed by changing the initial and final area of the orifice (for  $\zeta_1$  and  $\zeta_2$ ) and by changing the spring constant (for A and B). Some aircraft landing gear systems may be said to display somewhat similar behaviour.

Snowdon [1] has worked out the transient response of such a system for a set of parameters and indicated that the peak acceleration and displacement could be minimised employing the dual-phase damping characteristics. The excursions of the peak values of acceleration and displacement transmitted to the mass M for inputs like rounded step and rounded pulse displacements are sought by the authors for different combinations of values of A, B,  $\zeta_1$ ,  $\zeta_2$  (Fig. 1. 1 b) which information, it is felt, would be useful in the design of shock mounts.

# 2. SYSTEM EQUATIONS

The equation of motion for the system shown in Fig. 1. 1 a may be written as

$$M\ddot{Z}_1 + C\dot{Z}_1 + KZ_1 = C\dot{Z}_3 + KZ_3$$

with the initial conditions

 $\dot{Z}_1(o) = Z_1(o) = 0$ 

It is convenient to rewrite the equation as



where,

$$w_0^2 = \frac{K}{M}; \quad 2\zeta w_0 = \frac{C}{M}; \quad \zeta = \frac{C}{C_c} = \frac{C}{2\sqrt{KM}}$$

Dividing throughout by  $w_0^2 Z_{max}$  results in

$$\frac{\ddot{Z}_{1}}{w_{0}^{2}Z_{\max}} + 2\zeta \frac{\dot{Z}_{1}}{w_{0}Z_{\max}} + \frac{Z_{1}}{Z_{\max}} = 2\zeta \frac{\dot{Z}_{3}}{w_{0}Z_{\max}} + \frac{Z_{3}}{\dot{Z}_{\max}}$$

In the non-dimensional form, the above equation becomes

$$\ddot{Y} + 2\zeta \dot{Y} + Y = 2\zeta \dot{Z}_{3}' + Z_{3}'$$

where,

 $\dot{Y}(o) = 0 = Y(o)$  and  $Z_3'$ ,  $\dot{Z}_8'$  are respectively the non-dimensional displacement and velocity.

According to the characteristic shown in Fig. 1. 1 b,  $\zeta$  takes values as under

$$\begin{aligned} \zeta &= \zeta_1 \quad \text{where} \quad |\dot{Z}_3' - \dot{Y}| \leqslant A \\ &= \frac{\zeta_1 - \zeta_2}{A - B} (X - A) + \zeta_1; \quad |\dot{Z}_3' - \dot{Y}| \text{ lies between } A \text{ and } B \\ &\qquad X = |\dot{Z}_3' - \dot{Y}| \\ &= \zeta_3; \quad |\dot{Z}_3' - \dot{Y}| \geqslant B. \end{aligned}$$

Optimisation for the peak acceleration or displacement though could be done by gradient method, the authors prefer to use direct search because the latter furnishes information about the variations of the peak acceleration and displacement with the four parameters along with the optimum value.

The non-linear differential equation is solved by Runge-Kutta method using a digital computer IBM 360/44. The inputs considered are the rounded step and rounded pulse displacements (Fig. 2 a, b) for these possess finite first and second derivatives for all values of t including t = 0; application of these transients to the foundation of the simple system provides physically realistic values for the velocity and acceleration of both the foundation and the mounted mass M. The reference inputs are defined by

$$Z_3 = 0, \quad t < 0$$
  
=  $Z_{\max} [1 - e^{-\gamma w_s t} (1 + \gamma w_0 t)], \quad t \ge 0 \text{ (Rounded step)}$   
=  $Z_{\max} \frac{e^2}{4} (\gamma w_0 t)^2 e^{-\gamma w_s t}, \quad t \ge 0 \text{ (Rounded pulse)}$ 

where,

$$y = \frac{T}{2\tau} = \frac{\pi}{w_0 \tau}; \quad w_0 = \sqrt{\frac{K}{M}}$$

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The rise time  $\tau$  of the rounded step displacement is defined as the time required for the displacement to reach  $82\frac{9}{6}$  of its final value. The duration  $\tau$  of the displacement pulse is defined as the length of an equivalent rectangular pulse that has the same area  $(Z_{\max} e^2/2\gamma w_0)$  as that of the rounded pulse, but which is higher by  $17.6\frac{9}{6}$  than  $Z_{\max}$ .



FIG. 1-2

Range of parameters considered in the study for getting data are: A = 0(0.2)1

 $\gamma = 10$  defines mild shock and  $\gamma = 50$  defines severe shock. These two values are chosen for analysis.



FIG. 2

In the following section, peak acceleration refers to max  $|\ddot{Y}|$ , peak displacement to max |Y|.

# 3. RESULTS AND DISCUSSION

From the data computed, it is found that the peak acceleration and displacement show a variation with A, B,  $\zeta_1$ ,  $\zeta_2$  as typified in the accompanying graphs, Fig. 3. 1 through 3.32.

# 3.1 Input : Rounded Step

3.1.1 :  $\gamma = 10$  and  $\zeta_2 = 0.0$ 

The peak acceleration (i) increases with increase in  $\zeta_1$  for the values of  $A, B, \xi_1$ , less than or equal to 0.4, 0.6, 0.6 respectively, (ii) decreases and then increases with increase in  $\zeta_1$  in the range where A lies between 0.2 and 1.0 and B between 1.0 and 1.4, (iii) increases with increase in  $\zeta_1$  for values of B and A greater than 1.4 and 0.6 respectively. These variations are





shown in Fig. 3.1, for certain typical sets of parameters. The minimum value of the peak acceleration found in case (ii) is noted to decrease with increase of B and increase and then decrease with increase of A (Fig. 3.2).



The peak displacement (i) decreases with increase in  $\zeta_1$ . This decrement is found to be small for *B* less than 0.8 (Fig. 3.3); (ii) decreases with increase in *A* and *B* (Fig. 3.4, 3.5). Case (ii) is not true always. In many cases



it is found that the peak displacement increases with increase in A and B (for A greater than 0.6) for  $\zeta_1 \ge \zeta_c$  where  $\zeta_c$  itself varies from 0.2 to 0.5. But this increment is very small of the order of 1 in  $10^3$  or 1 in  $10^2$ .



 $3.1.2: \gamma = 10 \text{ and } \zeta_2 = 0.1$ 





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The peak acceleration (i) decreases with increase in  $\zeta_1$  for A equal to 0.2 and B less than or equal to 0.8. This decrement is very small (1 in 10<sup>3</sup>); (ii) decreases and then increases with an increase in  $\zeta_1$  for the values of A, B lying between 1.0 to 0.2 and 1.2 to 0.6 respectively. This decrement is very small; (iii) increases with increase in  $\zeta_1$  for values of B and A greater



than 1.2 and 0.8 respectively (Fig. 3.6). The minimum of the peak acceleration obtained in case, (ii) is found to decrease with an increase in *B* and increase and then decrease with an increase in *A* (Fig. 3.7). The peak acceleration increases with increase in  $\zeta_2$  (Fig. 3. 14 *a*).



FIG. 3.12 ROUNDED STEP Y = 50



The peak displacement (i) decreases with increase in  $\zeta_1$ . This decrement is small for *B* less than 0.8 (Fig. 3.3); (ii) decreases with increase in *B* and *A* 



FIG. 3.14 b ROUNDED STEP Y= 50







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(Figs. 3.4, 3.5). This is found to be violated for certain values of  $\zeta_1$ . The peak displacement increases with increase in *B* and *A* (for *A* greater than or equal to 0.6) for  $\zeta_1$  greater than a certain value which varies from 0.3 to 0.5. This increment is very small.



 $3.1.3: \gamma = 50$  and  $\zeta_2 = 0.0$ 





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The peak acceleration (i) increases and then decreases with increase in  $\zeta_1$  for A and B lying between 0.4 and 0.0, 0.8 and 0.2 respectively; (ii) decreases with increase in  $\zeta_1$  in the range where A lies between 1.0 to 0.0 and B between 1.8 to 1.0; (iii) increases with increase in  $\zeta_1$  for A = 1.0



FIG. 3.26 ROUNDED PULSE 1 = 10





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The peak displacement (i) decreases with increase in  $\zeta_1$ ; (ii) decreases, with increase in *B* for *A* less than or equal to 0.6 and beyond this range it remains a constant; (iii) decreases with increase in *A* (Fig. 3.10, 3.11, 3.12).

 $3.1.4: \gamma = 50 \text{ and } \zeta_2 = 0.1$ 

The peak acceleration (i) decreases with increase in  $\zeta_1$  for A and B lying between 0.2 to 1.0 and 0.4 to 1.8 respectively; (ii) remains constant with A and B for a constant  $\zeta_1$  (Figs. 3.13, 3.9).

The peak acceleration increases with increase in  $\zeta_2$  (3.14 b).

The peak displacement (i) decreases with increase in  $\zeta_1$ ; (ii) decreases with increase in B for A less than or equal to 0.6 and remains constant with B for values of A greater than 0.6; (iii) decreases with increase in A (Figs. 3.15, 3.11, 3.12).

The peak displacement increases with increase in  $\zeta_2$  (Fig. 3.16).

#### 3.2 Input: Rounded Pulse

 $3.2.1: \gamma = 10 \text{ and } \zeta_2 = 0.0$ 

The peak acceleration (i) decreases with increase in  $\zeta_1$  for  $\zeta_1$  less than or equal to 0.8 and A, B in the ranges 0 to 0.8 and 0 to 1.2 respectively. This decrement is very small, 1 in 10<sup>3</sup>; (ii) decreases and then increases with increase in  $\zeta_1$  for A greater than or equal to 0.6 and B greater than or equal to 1.4; (iii) remains almost constant with A and B for the ranges 0 to 0.8 and 0 to 1.2 respectively (Figs. 3.17, 3.18, 3.19).

The peak displacement (i) decreases with increase in  $\zeta_1$ ; (ii) decreases with increase in A and B; (iii) cases (i) and (ii) hold good for  $\zeta_2 = 0.0$  and 0.1 (See Figs. 3.20, 3.21, 3.22).

 $3.2.2: \gamma = 10 \text{ and } \zeta_2 = 0.1$ 

The peak acceleration (i) decreases with increase in  $\zeta_1$  for A and B in the ranges 0 to 1.0 and 0 to 1.6 respectively; (ii) remains constant with A and B for the values of A and B lying between 0 to 1.0 and 0 to 1.6 respectively; (iii) decreases with increase in B for A greater than 0.8 and B greater than 1.6 (Figs. 3.23, 3.24, 3.25).

The peak acceleration increases with increase in  $\zeta_2$  (Fig. 3.26). The peak displacement increases with increase in  $\zeta_2$  (Fig. 3.27),









 $3.2.3: \gamma = 50 \text{ and } \zeta_2 = 0.0, 0.1$ 

The peak acceleration (i) decreases with increase in  $\zeta_1$ . The decrement is very small; (ii) remains constant with A and B; (iii) increases with increase in  $\zeta_2$  (Figs. 3.28, 3.29).

The peak displacement (i) increases with increase in  $\zeta_1$ ; (ii) decreases with increase in A and B; (iii) decreases with increase in  $\zeta_2$  (Figs. 3.30, 3.31\*, 3.32,\* 3.27).

#### 4. ILLUSTRATION AND COMPARISON

The values of the peak acceleration and displacement obtained with dual-phase damping are compared with that of constant damping. Examples are given.

### 4.1 Rounded Step Displacement

 $4.1.1 : \gamma = 10$ 

The minimum value obtained for the peak acceleration is 0.8631. The values of the parameters are A = 1.0, B = 1.4,  $\zeta_1 = 0.4$  and  $\zeta_2 = 0.0$ . The peak displacement for the above set is 1.2636. For  $\zeta_1 \ge 0.7$ , the peak displacement in most cases is approximately 1.0. But the peak acceleration is different and greater than 0.86.

 $4.1.2: \gamma = 50$ 

The minimum peak acceleration got is 0.9528, with peak displacement approximately 1.0 for  $\zeta_1 = 0.9$ ,  $\zeta_2 = 0.0$  and A in the range 0 to 1.0 and B 1.0 to 1.8.

4.2 Rounded Pulse

 $4.2.1: \gamma = 10$ 

The minimum peak acceleration in this case is 0.8010 with a peak displacement 0.3093. The values of the parameters A, B,  $\zeta_1$ ,  $\zeta_2$  are 0.2, 0.4, 0.9 and 0.0 respectively.

The minimum peak displacement is 0.1207 with peak acceleration 1.69 for A = 1.0, B = 2.0,  $\zeta_1 = 0.9$  and  $\zeta_2 = 0.0$ .

<sup>\*(</sup>A) indicates the scale marked for the curve with constant A; similarly for (B),

But, an optimum case could be chosen to be the one where the peak acceleration and displacement are 0.8017 and 0.1951 respectively. The values of the parameters are A = 0.8, B = 1.0,  $\zeta_1 = 0.9$  and  $\zeta_2 = 0.0$ .



 $4.2.2: \gamma = 50$ 

The minimum value of the peak acceleration obtained is 0.9937 and the peak displacement is about 0.08. The parameter values are A = 1.0B = 2.0,  $\zeta_1 = 0.9$  and  $\zeta_2 = 0.0$ .

#### 4.3 Comparison with Constant Damping

Considering the first case (*i.e.*,  $\gamma = 10$ , step displacement) with constant damping  $\zeta = 0.4$ , the peak acceleration of the mass M [2] is nearly 3.0 and the displacement is about 1.5. For low values of damping ( $\zeta \leq 0.05$ ) the peak acceleration is found to be less than 1.0 and the peak displacement is nearly 2.0. By using the dual-phase damping, it is possible to achieve low values of peak acceleration and displacement, *i.e.*, 0.8631 and 1.2636 respectively.

A similar behaviour is found in other cases also.

#### 5. CONCLUSION

5.1. For all the cases considered here, the peak acceleration increases with increase in the value of  $\zeta_2$ . The variation with  $\zeta_1$  is not much compared to that with  $\zeta_2$ . It depends on the value of  $\gamma$ . If  $\gamma = 50$ , the variation is not much. If  $\gamma = 10$ , the peak acceleration decreases and then increases with increase in  $\zeta_1$ , for many cases. This is observed for both inputs.

Irrespective of the type of input, the variation of peak acceleration with A and B is negligible for  $\gamma = 50$ .

5.2. The peak displacement increases with increase in  $\zeta_2$  for rounded step displacement for one set of A, B,  $\zeta_1$  and it decreases with increase in  $\zeta_1$  for a set of A, B,  $\zeta_2$ .

In the case of rounded pulse displacement, for  $\gamma = 10$ , the peak displacement increases with increase in  $\zeta_2$  and for  $\gamma = 50$ , it decreases with increase in  $\zeta_2$ .

The peak displacement, for the pulse case, decreases with increase in  $\zeta_1$  for  $\gamma = 10$  and increases with increase in  $\zeta_1$  for  $\gamma = 50$ .

5.3. With dual-phase damping, the peak acceleration and displacement of the mass M could be reduced. By suitably selecting the parameters, the system could be optimised for either the peak acceleration or displacement keeping one or the other within bounds,

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5.4. The effect of applying dual-phase damping to an aircraft landing gear is being investigated by the authors.

#### ACKNOWLEDGEMENT

The authors are most grateful to Dr. R. Sankaranarayanan, Manager, Computer Services, H.A.L. and Dr. V. T. Nagaraj, Design Engineer, H.A.L., Bangalore, for their interest and support in carrying out the work.

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