Short Communication

A note on the thermal buckling of a thin annular circular plate of variable thickness

P. K. JHA

Department of Physics, Government College of Engineering and Technology, Raipur 492 002, M.P.

Received on January 30, 1978

Abstract

In this note, the buckling of a heated annular circular plate of thickness varying as an exponential function of radial distance is considered. The general stability criterion has been derived.

Ley words : Thermal buckling, Whittaker functions, annular plate, stability criterion.

1. Introduction

The literature on thermal buckling of plates is rare. Nowacki⁴ has studied the buckling of a heated rectangular plate under various boundary conditions. The buckling and curling of a heated thin circular plate of constant thickness has been investigated by Mansfield³ when the temperature varies through the thickness and the edges are restrained. A similar problem for the buckling of simply supported plate under symmetrical temperature distribution has been described by Klosner and Forry² by using Rayleigh-Ritz method. Sarkar⁵ has discussed the thermal buckling of a circular plate taking its thickness to be constant.

This paper records the discussion of a buckled annular circular plate when its thickness varies as the exponential function of radial distance under stationary temperature distribution. The edge of the plate is restrained. The general stability criterion has been obtained from which the critical compression and critical temperature can be determined for different values of parameters involved.

2. Statement of the problem and fundamental equations Let us consider an isotropic thin annular circular plate of variable thickness $h = h_0 e^{-qr}$, h_0 and q being constants and r the radial distance. The plate is supposed to occupy 129

P. K. JHA

the space $-h/2 \le z \le h/2$ and is subjected to stationary temperature distribution. We further assume that the edge of the plate is restrained so that the displacement in the plane of the plate is zero. The deflections are supposed to be governed by the linear theory of thermo-elasticity. The compression in the middle plane of the plate is assume to be $N_r = N_0 e^{-2ar}$.

The equilibrium equation relating the components of moment for the problem?

$$\left(M_{r} + \frac{dM_{r}}{dr} \cdot dr\right)(r + dr) d\theta - M_{r} \cdot rd\theta - M_{\theta} \cdot drd\theta + hN_{r}\phi rd\theta dr = 0$$
(2.)

where d is the angle subtended between the two radial planes and

$$M_r = D\left(\frac{d\phi}{dr} + \frac{v}{r}\phi\right) - \frac{M_T}{1 - v}$$

$$M_\theta = D\left(\frac{vd\phi}{dr} + \frac{\phi}{r}\right) - \frac{M_T}{1 - v}$$

$$M_T = \alpha E \int_{-h/2}^{+h/2} Tzdz$$

130

$$\dot{N}_{r} = K \left(\frac{dU}{dr} - \frac{v}{r} U \right) - \frac{N_{T}}{1 - v} = N_{0} e^{-2ar}$$

$$N_{T} = \alpha E \int_{-N/2}^{+N/2} T dz$$

$$K = \frac{Eh}{(1 - v^{2})}, \quad h = h_{0} e^{-ar}, \quad D = D_{0} e^{-3ar}$$
where

$$D_0 = \frac{Eh_0^3}{12(1-v^2)} \, . \qquad .$$

(2.]

<u>(</u>2.1

(2.

Neglecting higher order quantities, equation (2.1) becomes

$$M_{r} + r \frac{dM_{r}}{dr} - M_{\theta} + h N_{r} \phi r = 0.$$

.

Substituting values of M_r , M_{θ} , h, D and N_r in (2.3) we get

$$\frac{r^{2} d^{2} \phi}{dr^{2}} + (1 - 3qr) \frac{r d\phi}{dr} + \left(\frac{h_{0}N_{0}}{D_{0}}r^{2} - 3vq^{r-1}\right)\phi$$

$$= \frac{r^{2} e^{3er} \frac{dM_{r}}{dr}}{D_{0}(1 - v)}$$

Putting $\rho = r/a$, $\beta = b/a$ the outer and inner boundaries correspond to $\rho = 1$ and $\rho = \beta$. Then (2.4) becomes $\rho^2 \frac{d^2 \phi}{d\rho^2} + (1 - k\rho) \rho \frac{d\phi}{d\rho} + (\lambda \rho^2 - \nu k \rho - 1) \phi$ $= \frac{a \rho^2 e^{k\rho} \frac{dM_T}{d\rho}}{D_0 (1 - \nu)}$ (2.5)

where

$$k = 3aq$$
, $\lambda = \frac{h_0 N_0 a^2}{D_0}$ are constants.

We further suppose that the outer boundary of the plate is clamped and supported and the inner boundary is clamped. Then the appropriate boundary conditions of the problem are¹

$$\phi = 0$$
 when $\rho = 1$, $\rho = \beta$
 $\omega = 0$ when $\rho = 1$. (2.6)

• • • • • • • •

3. Solution of the problem

Let the plate be subjected to the temperature field

 $T(r, z) = \tau_0(r) + z \tau(r).$ (3.1)

For a plate of medium thickness buckling occurs when $\tau = 0$, $\tau_0 \neq 0$. Therefore

 $M_T = 0$ and from (2.5) we get

$$\rho^2 \frac{d^2 \phi}{d\rho^2} + (1 - k\rho) \rho \frac{d\phi}{d\rho} + (\lambda \rho^2 - \nu k\rho - 1) \phi = 0.$$
(3.2)

Putting $X = k\rho$, $\phi = \psi e^{\chi/2}$ the above equation becomes

$$X^{2}\frac{d^{2}\psi}{dX^{2}} + X\frac{d\psi}{dX} + \left\{ \left(\frac{1}{2} - \nu\right)X + \left(\frac{\lambda}{k^{2}} - \frac{1}{4}\right)X^{2} - 1 \right\} \psi = 0.$$
(3.3)

Now writing $\psi = X^{-1/2} \xi$ in equation (3.3) we obtain

$$4X^{2}\frac{d^{2}\xi}{dX^{2}} + \left\{-3 + 2\left(1 - 2\nu\right)X - \left(1 - \frac{4\lambda}{k^{2}}\right)X^{2}\right\}\xi = 0$$
(3.4)

Again, putting

$$\sqrt{1-\frac{4\lambda}{k^2}} X = \eta, \qquad \frac{1-2\nu}{2\sqrt{-1\frac{4\lambda}{k^2}}} = \epsilon$$

in (3.4) we get 8 $\frac{d^2 \xi}{dn^2} + \left\{ -\frac{1}{4} + \frac{\epsilon}{n} + \frac{\frac{1}{4} - 1}{n^2} \right\} \xi = 0.$ (3.5) The solution of (3.5) is given by $\xi = AW_{\epsilon,1}(\eta) + BW_{-\epsilon,1}(-\eta)$

(3.6)

(3.7)

Notes

where $W(\eta)$ are Whittaker functions⁶ and A, B are constants. Hence

 $\phi = (k\rho)^{-1/2} e^{k\rho/2} \left[AW_{\epsilon,1} (y\rho) + BW_{-\epsilon,1} (-y\rho) \right]$

where

$$y = \sqrt{1 - \frac{4\lambda}{k^2}}$$

since

$$a\phi = -\frac{d\omega}{d\rho}$$

we obtain

 $\omega = AF_1(\rho) + BF_2(\rho) + C$ (3.8)

where $F_1(\rho)$ and $F_2(\rho)$ are functions of ρ obtained after integration and C is a constant of integration. Introducing the boundary conditions (2.6) in (3.8) and eliminating constants A, B, C we get

The determinant on expansion yields

$$W_{\epsilon,1}(y) W_{-\epsilon,1}(-y\beta) - W_{-\epsilon,1}(-y) W_{\epsilon,1}(y\beta) = 0.$$
^(3.10)

This is the general stability criterion from which the critical compression and critical temperature can be determined for different values of the parameters if the roots of equation (3.10) are known.

4. Nomenclature

$$\omega$$
 = Deflection of the middle surface

$$\dot{\phi} = -\frac{d\omega}{dr}$$
 = Slope of the middle surface

$$D = \frac{Eh^3}{12(1-v^2)} = \text{Flexural rigidity}$$

$$\alpha$$
 = Coefficient of thermal expansion

$$E, v =$$
 Young's modulus and Poisson's ratio

$$M_{e_1}M_{\theta}$$
 = Components of moment

- N_r = Compression in the middle plane of the plate, a variable quantity
- a, b = Radii of the outer and inner boundaries of the plate

$$\beta = \frac{a}{b}$$

5. Acknowledgement

The author would like to express his appreciation to Dr. M. P. Pateria, Government Engineering College, Raipur, for his help and encouragement during the present work.

References

1.	CONWAY, H. D.	The bending of symmetically loaded circular plates of variable thickness, Jour. Appl. Mech. (Trans. ASME), 1948, 15, 1.
2.	Klosner, J. M. and Forray, M. J.	Buckling of simply supported plates under arbitrary symmetrical temperature distributions, Jour. Aero-sp. Sci., 1958, 25, 181.
3.	MANSFIELD, E. H.	Proc. Royal Society Series-A, 1962, No. 1334, 286, 316.

NOWACKI, W. Thermo-Elasticity (International series of monographs in aero-nautics and astronautics), Addison Wesley, 1962, p. 486.
 SARKAR, S. K. Ind. Journ. of Mech. and Maths., 1967, 16 (2), 53.
 SLATER, L. J. Confluent Hypergeometric Functions, Cambridge University Press, 1960. p. 9.
 TIMOSHENKO AND Theory of Plates and Shells, McGraw-Hill Book Co. Ltd., 1959, p. 53.